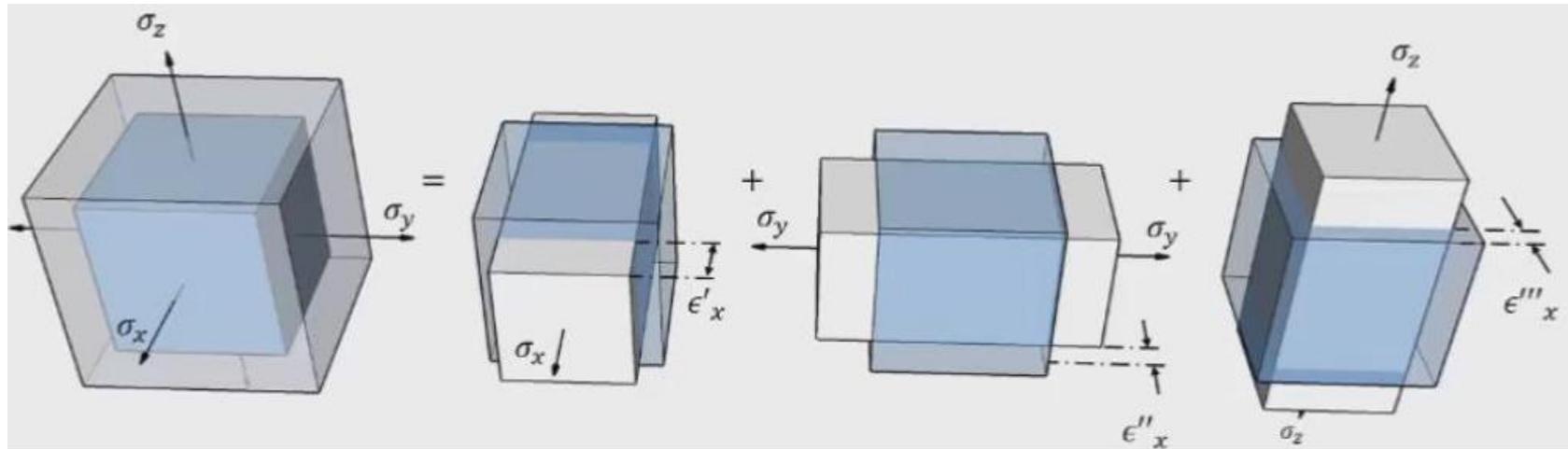
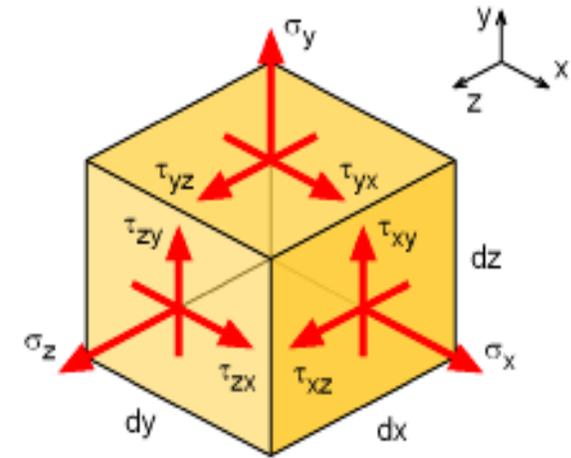


Complex Stresses

- The most general state of stresses is the **complex stresses** represented by the unit cube shown here.
- It is assumed that shear stresses can generate only shear strains. Thus, the longitudinal strains are produced exclusively by the normal stresses.



$$\epsilon'_x = \frac{\sigma_x}{E}$$

$$\epsilon''_x = -\nu \frac{\sigma_y}{E}$$

$$\epsilon'''_x = -\nu \frac{\sigma_z}{E}$$

Hooke's law:

$$\epsilon = \frac{\sigma_x}{E}$$

Poisson's ratio:

$$\nu = -\frac{\epsilon_{lat}}{\epsilon_{log}}$$

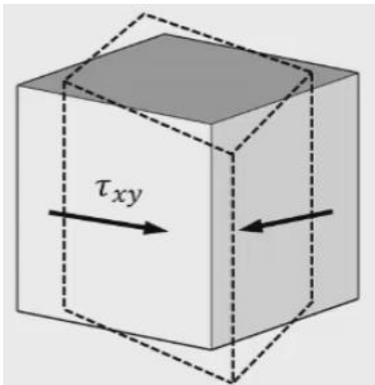
Complex Strains

$$\varepsilon_x = \varepsilon'_x + \varepsilon''_x + \varepsilon'''_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E} \quad \rightarrow \quad \varepsilon_x = \frac{1}{E} [\sigma_x - \nu(\sigma_y + \sigma_z)]$$

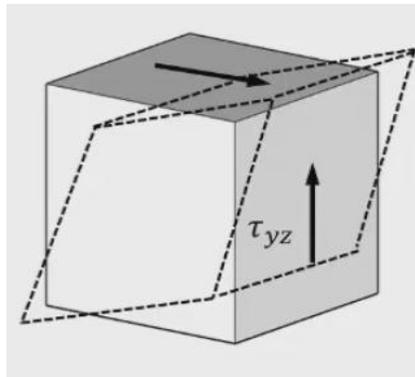
Using the same procedure results in

$$\varepsilon_y = \frac{1}{E} [\sigma_y - \nu(\sigma_x + \sigma_z)] \quad \text{and} \quad \varepsilon_z = \frac{1}{E} [\sigma_z - \nu(\sigma_x + \sigma_y)]$$

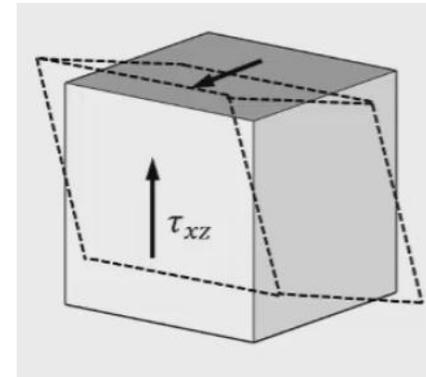
- ✓ Shear strains cause deformation only, and will not cause other strains in the material.



$$\gamma_{xy} = \frac{1}{G} \tau_{xy}$$

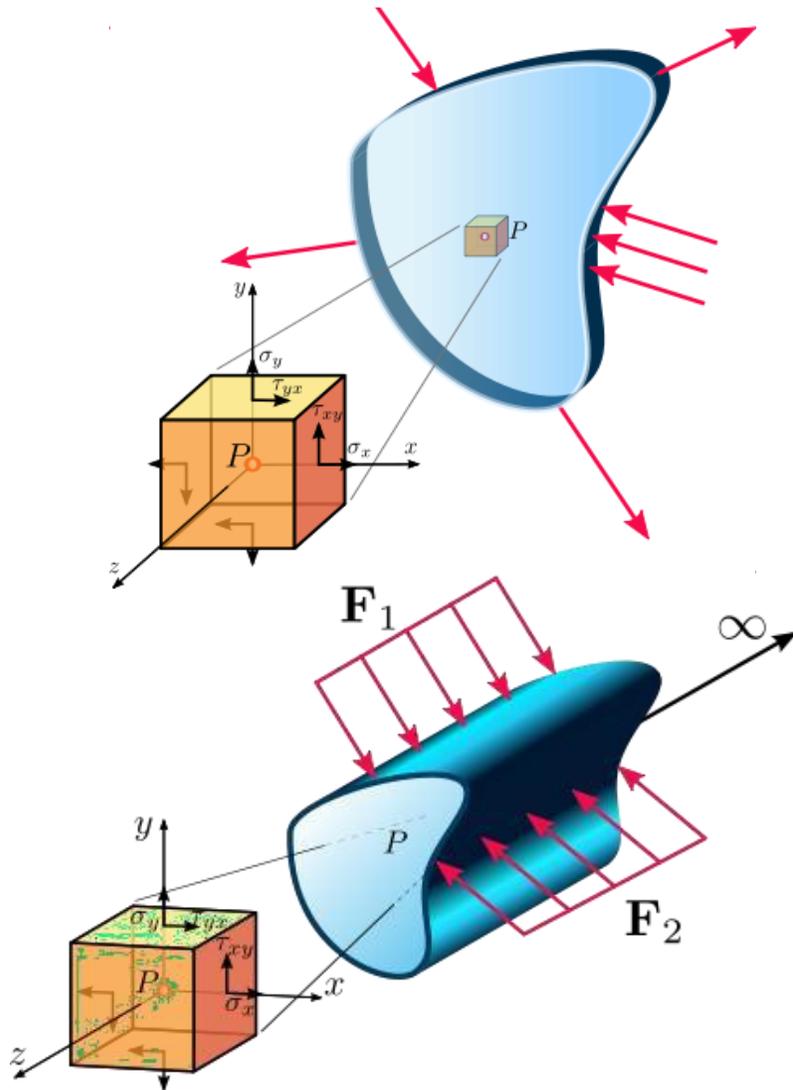


$$\gamma_{yz} = \frac{1}{G} \tau_{yz}$$



$$\gamma_{xz} = \frac{1}{G} \tau_{xz}$$

Complex Strains

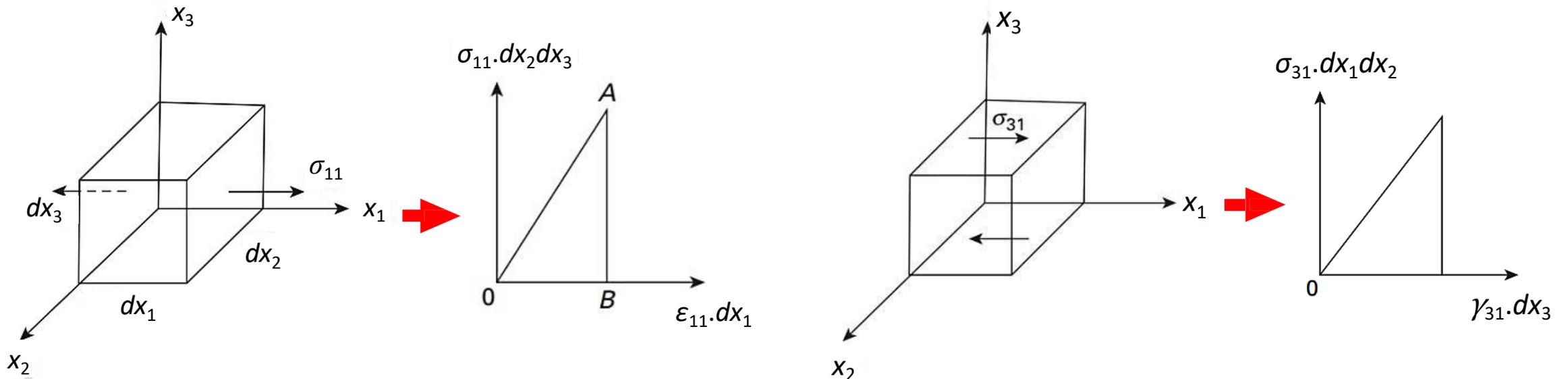


- In sheets and plates, the state of stress can be assumed to be bidimensional because normal stresses (normal to the surface) are **zero** at the surface. This state of stress is also known as **plane stress**.
- If **one dimension is infinite** with respect to the other two, strain in it is constrained zero; hence, one has two dimensions left. This state is called bidimensional or, more commonly, **plane strain**.
- Plane strain occurs when strain is constrained zero in one direction by some other means.

Strain energy (Deformation energy)

- The **work done** (W) is converted into **heat** (Q) and an increase in **internal energy** (U).
- For most solids, the elastic work produces an **insignificant** amount of heat. Hence, the **work done** on a body during deformation is **converted** into **internal energy** (strain energy).
- The **work done** is given by product of **force** and **change in length** (see the figure).

$$W = \frac{1}{2} [\sigma_{11}(dx_2 dx_3) \epsilon_{11} dx_1] = \frac{1}{2} [\sigma_{11} \epsilon_{11} (dx_1 dx_2 dx_3)]$$



Strain energy (Deformation energy)

The **work done** (W) per unit volume is:

$$W_{11} = \frac{1}{2} \sigma_{11} \varepsilon_{11}$$

- for the **shear stress**, σ_{31} , the work done per unit volume is:

$$W_{31} = \frac{1}{2} \sigma_{31} \varepsilon_{31}$$

- for **two or more stresses**, the total work done per unit volume or the strain energy density is:

$$W = \frac{1}{2} [\sigma_{11} \varepsilon_{11} + \sigma_{22} \varepsilon_{22} + \sigma_{33} \varepsilon_{33} + 2\sigma_{12} \gamma_{12} + 2\sigma_{23} \gamma_{23} + 2\sigma_{31} \gamma_{31}]$$

- In more compact indicial notation, we can write;

$$W = \frac{1}{2} \sigma_{ij} \varepsilon_{ij}$$

- For a linearly elastic solid under a **uniaxial stress** we can use the **Hooke's law** to obtain;

$$U = W = \frac{1}{2E} \sigma_{ij}^2$$

Anisotropic Effects

- Real materials are **never** perfectly isotropic
- In highly **anisotropic** materials, any one component of stress can cause strains in all six components.
- In the general 3-D, there are nine components of stress and a corresponding nine components of **strain**.

Stress components

$$\begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{12} & \sigma_{22} & \sigma_{23} \\ \sigma_{13} & \sigma_{23} & \sigma_{33} \end{bmatrix}$$

Strain components

$$\begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{bmatrix} = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{12} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{13} & \epsilon_{23} & \epsilon_{33} \end{bmatrix}$$

- The off-diagonal components are equal, i.e., $\sigma_{13} = \sigma_{31}$, $\sigma_{12} = \sigma_{21}$ and $\sigma_{23} = \sigma_{32}$ as well as $\epsilon_{13} = \epsilon_{31}$, $\epsilon_{12} = \epsilon_{21}$ and $\epsilon_{23} = \epsilon_{32}$ as can be seen above.

Anisotropic Effects

- The indices can be replaced into;

$$11 \Leftrightarrow 1 \quad 23 \Leftrightarrow 4$$

$$22 \Leftrightarrow 2 \quad 13 \Leftrightarrow 5$$

$$33 \Leftrightarrow 3 \quad 12 \Leftrightarrow 6$$

therefore we have

$$\begin{bmatrix} \sigma_1 & \sigma_6 & \sigma_5 \\ \sigma_6 & \sigma_2 & \sigma_4 \\ \sigma_5 & \sigma_4 & \sigma_3 \end{bmatrix} \text{ and } \begin{bmatrix} \varepsilon_1 & \varepsilon_6/2 & \varepsilon_5/2 \\ \varepsilon_6/2 & \varepsilon_2 & \varepsilon_4/2 \\ \varepsilon_5/2 & \varepsilon_4/2 & \varepsilon_3 \end{bmatrix}$$

Because we have

$$\varepsilon_4 = 2\varepsilon_{23} = \gamma_{23}$$

$$\varepsilon_5 = 2\varepsilon_{13} = \gamma_{13} \quad \text{why ?}$$

$$\varepsilon_6 = 2\varepsilon_{12} = \gamma_{12}$$

- For the generalized case, Hooke's law may be expressed as:

$$\sigma_i = C_{ij} \cdot \varepsilon_j$$

$$\varepsilon_i = S_{ij} \cdot \sigma_j$$

Anisotropic Effects

- In matrix format, the stress-strain relation showing the **36** (6 x 6) independent components of stiffness

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & C_{14} & C_{15} & C_{16} \\ C_{21} & C_{22} & C_{23} & C_{24} & C_{25} & C_{26} \\ C_{31} & C_{32} & C_{33} & C_{34} & C_{35} & C_{36} \\ C_{41} & C_{42} & C_{43} & C_{44} & C_{45} & C_{46} \\ C_{51} & C_{52} & C_{53} & C_{54} & C_{55} & C_{56} \\ C_{61} & C_{62} & C_{63} & C_{64} & C_{65} & C_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{bmatrix}$$

- In matrix format, the stress-strain relation showing the 36 (6 x 6) independent components of stiffness, which means $C_{ij} = C_{ji}$ and $S_{ij} = S_{ji}$
- Therefore, the 36 components (6x6) for the general **anisotropic** linear elastic solid are reduced to **21**. In **Isotropic** case, the elastic constants are reduced from **21** to **2**.

Anisotropic Effects

$$\begin{pmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{pmatrix} = \begin{pmatrix} C_{11} & C_{12} & C_{13} \\ & C_{22} & C_{23} \\ & & C_{33} \\ & & & C_{44} \\ & & & & C_{55} \\ & & & & & C_{66} \end{pmatrix} \begin{pmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{33} \\ \gamma_{23} \\ \gamma_{13} \\ \gamma_{12} \end{pmatrix}$$

Shear decoupling → normal stress × shear strains
 Shear strain in a plane can't be due to shear stress in another plane

4 $\frac{360}{4} = 90^\circ$ retain symmetry

Crystal Structure	Rotational Symmetry	Independent Elastic constants	Diagram	n-Fold means rotation of (360/n) retains the symmetry
Triclinic	None	21		$a \neq b \neq c$ and $\alpha \neq \beta \neq \gamma$
Orthorhombic	2 perpendicular 2fold rotations axis	9		$a \neq b \neq c$ and $\alpha = \beta = \gamma = 90^\circ$ $\sigma_{11} C_{11} \epsilon_{11} \quad \sigma_{22} C_{22} \epsilon_{22}$
Cubic	4 Three-fold	3		$a = b = c$ and $\alpha = \beta = \gamma = 90^\circ$

$C_{11} = C_{22} = C_{33} \quad C_{44} = C_{55} = C_{66} \quad C_{23} = C_{13} = C_{12}$

Anisotropic Effects

- The number of independent elastic constants in a cubic system is three.
- For **isotropic** materials there are two independent constants

$$\begin{bmatrix} C_{11} & C_{12} & C_{12} & 0 & 0 & 0 \\ \cdot & C_{11} & C_{12} & 0 & 0 & 0 \\ \cdot & \cdot & C_{11} & 0 & 0 & 0 \\ \cdot & \cdot & \cdot & C_{44} & 0 & 0 \\ \cdot & \cdot & \cdot & \cdot & C_{44} & 0 \\ \cdot & \cdot & \cdot & \cdot & \cdot & C_{44} \end{bmatrix}$$

because:

$$C_{44} = \frac{C_{11} - C_{12}}{2}$$

where:

$$C_{11} = \frac{E(1 - \nu)}{(1 + \nu)(1 - 2\nu)} \quad C_{12} = \frac{E\nu}{(1 + \nu)(1 - 2\nu)} \quad C_{44} = G = \frac{E}{2(1 + \nu)}$$

- For **anisotropic** systems, the equation of C_{44} does not apply, and we define an anisotropy

ratio:

$$A = \frac{2C_{44}}{C_{11} - C_{12}} \neq 1$$

- For the elastic compliances, we have, for the anisotropic case

$$\begin{bmatrix} S_{11} & S_{12} & S_{12} & 0 & 0 & 0 \\ \cdot & S_{11} & S_{12} & 0 & 0 & 0 \\ \cdot & \cdot & S_{11} & 0 & 0 & 0 \\ \cdot & \cdot & \cdot & S_{44} & 0 & 0 \\ \cdot & \cdot & \cdot & \cdot & S_{44} & 0 \\ \cdot & \cdot & \cdot & \cdot & \cdot & S_{44} \end{bmatrix}$$

- For isotropic materials

$$S_{44} = 2(S_{11} - S_{12})$$

Anisotropic Effects

- Young's modulus:

$$E = \frac{1}{S_{11}}$$

- Rigidity or shear modulus:

$$G = \frac{1}{2(S_{11} - S_{12})} = \frac{1}{S_4}$$

- Compressibility (B) and bulk modulus (K):

$$B = \frac{1}{K} = \frac{\varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33}}{-\frac{1}{3}(\sigma_{11} + \sigma_{12} + \sigma_{13})}$$

- Poisson's ratio:

$$\nu = -\frac{S_{12}}{S_{11}}$$

- Lamé's constants:

$$\mu = C_{44} = \frac{1}{2}(C_{11} - C_{12}) = \frac{1}{S_{44}} = G$$

$$\lambda = C_{12} = \frac{E\nu}{(1 + \nu)(1 - 2\nu)}$$

Expressing the **strains as function** of stresses for isotropic materials

$$\varepsilon_1 = S_{11}\sigma_1 + S_{12}\sigma_2 + S_{12}\sigma_3 = \frac{1}{E} [\sigma_1 - \nu(\sigma_2 + \sigma_3)]$$

$$\varepsilon_2 = S_{12}\sigma_1 + S_{11}\sigma_2 + S_{12}\sigma_3 = \frac{1}{E} [\sigma_2 - \nu(\sigma_1 + \sigma_3)]$$

$$\varepsilon_3 = S_{12}\sigma_1 + S_{12}\sigma_2 + S_{11}\sigma_3 = \frac{1}{E} [\sigma_3 - \nu(\sigma_1 + \sigma_2)]$$

$$\varepsilon_4 = \frac{1}{2} (S_{11} - S_{12})\sigma_4 = \frac{1}{G} \sigma_4$$

$$\varepsilon_5 = \frac{1}{2} (S_{11} - S_{12})\sigma_5 = \frac{1}{G} \sigma_5$$

$$\varepsilon_6 = \frac{1}{2} (S_{11} - S_{12})\sigma_6 = \frac{1}{G} \sigma_6$$

Anisotropic Effects

Expressing the **stresses as function** of strains for isotropic materials

$$\sigma_1 = C_{11}\varepsilon_1 + C_{12}\varepsilon_2 + C_{12}\varepsilon_3 = (2\mu + \lambda)\varepsilon_1 + \lambda\varepsilon_2 + \lambda\varepsilon_3$$

$$\sigma_2 = C_{12}\varepsilon_1 + C_{11}\varepsilon_2 + C_{12}\varepsilon_3 = \lambda\varepsilon_1 + (2\mu + \lambda)\varepsilon_2 + \lambda\varepsilon_3$$

$$\sigma_3 = C_{12}\varepsilon_1 + C_{12}\varepsilon_2 + C_{11}\varepsilon_3 = \lambda\varepsilon_1 + \lambda\varepsilon_2 + (2\mu + \lambda)\varepsilon_3$$

$$\sigma_4 = \frac{1}{2}(C_{11} - C_{12})\varepsilon_4 = \mu\varepsilon_4$$

$$\sigma_5 = \frac{1}{2}(C_{11} - C_{12})\varepsilon_5 = \mu\varepsilon_5$$

$$\sigma_6 = \frac{1}{2}(C_{11} - C_{12})\varepsilon_6 = \mu\varepsilon_6$$

Elastic Properties of Ceramics

- **Porosity** is the crucial parameters on which the elastic modulus of ceramics strongly depends
- Wachtman and MacKenzie expression for the elastic modulus of ceramics as a **function of porosity** is:

$$E = E_o (1 - f_1 p - f_2 p^2)$$

are constants f_2 and f_1 is the porosity and p where

- For **spherical voids** and For relatively **low porosity**,

$$E = E_o (1 - 1.9p)$$

- The presence of **microcracks**, which affect the elastic properties of ceramics, **decrease** the stored elastic energy and **reduce** the effective Young's modulus.

Elastic Properties of Ceramics

- The expression developed by Salganik explains how the presence of microcracks change the Young modulus of ceramics;

$$\frac{E}{E_0} = \left[1 + \frac{16(10 - 3\nu_0^2)(1 - \nu_0^2)}{45(2 - \nu_0)} Na^3 \right]^{-1} = (1 + ANa^3)^{-1}$$

where a is the radius of a mean crack, and N is the number of cracks per unit volume.

- O'Connell and Budiansky arrived at a slightly different expression

$$\frac{E}{E_0} = 1 - \frac{16(10 - 3\nu)(1 - \nu^2)}{45(2 - \nu)} f_s$$

- where f_s is the volume fraction of cracks and ν is Poisson's ratio of the porous material

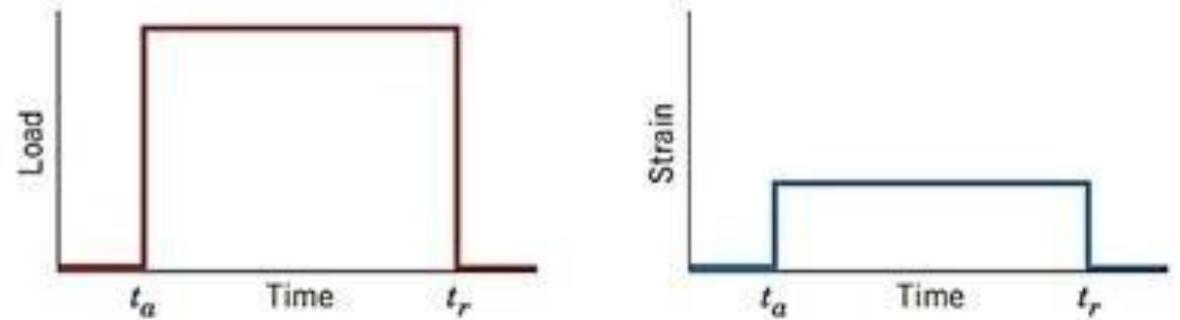
$$\nu = \nu_0 \left(1 + \frac{16f_s}{9} \right)$$

Elastic Properties of Polymers

- Elastic behaviour of polymeric materials is strongly dependent on both **temperature** and **time**.
- The behaviour is called viscoelastic or anelastic.
- In most polymers, there are dramatic changes in E between 20° C and 200°C, for most metals and ceramics, the changes in E in this range can be neglected.
- **Above** glass transition temperature T_g , E is considerably **low**, and the behaviour of the polymer can be described as rubbery and viscous. **Below** T_g , E is considerably **higher**, and the behaviour is closer to **linear elastic**.

Anelasticity (Viscoelastic behaviour)

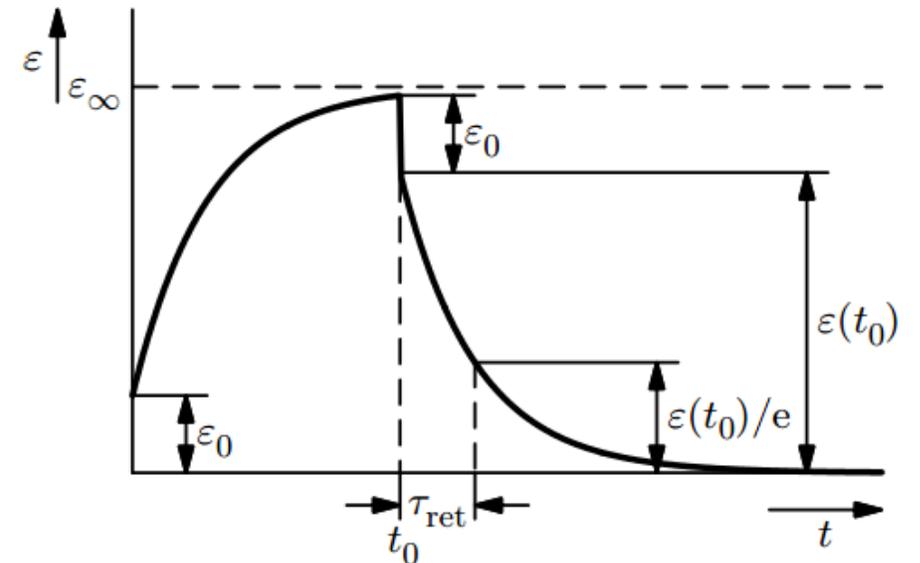
- Elastic deformation is time independent, and upon release of the load the strain is totally recovered (see the figure.)



- If the stress in a polymer is raised abruptly from zero to a value σ , the strain increases instantaneously to a value ϵ_0 (as time independent case), but then it further increases with time.

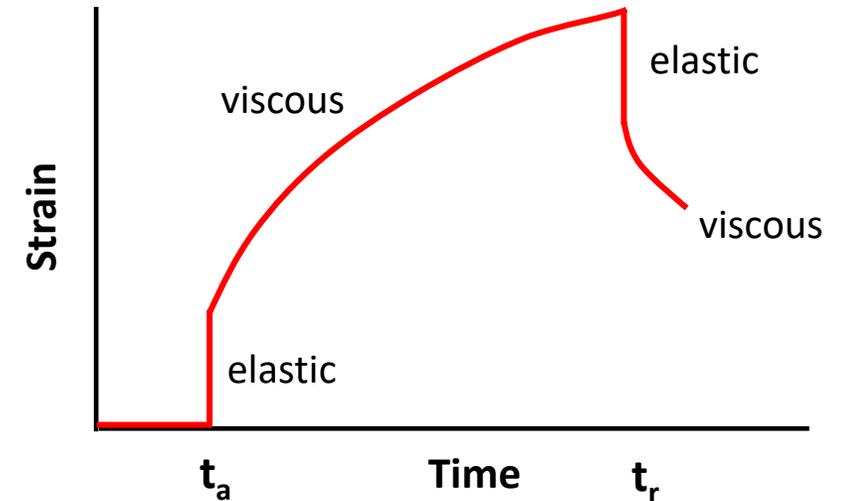
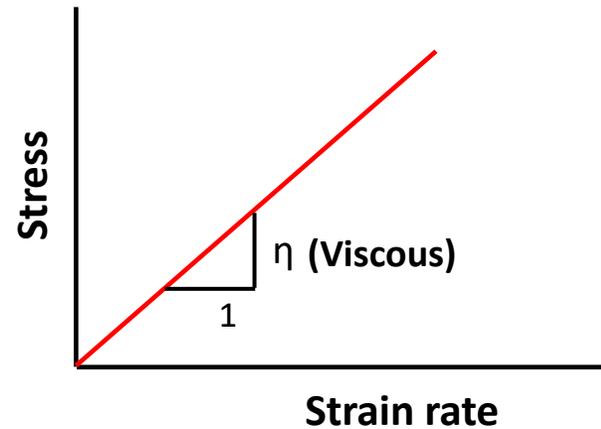
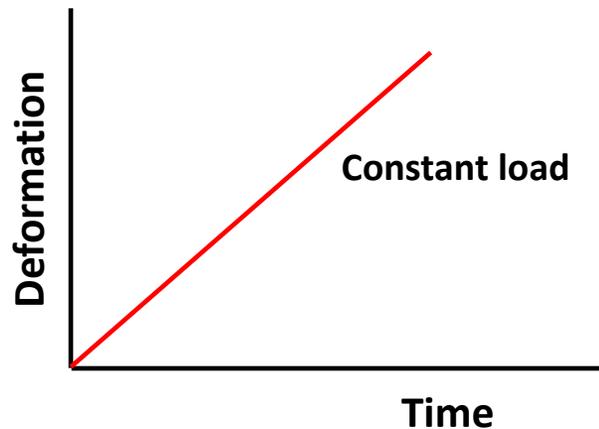
- the time-dependent Young's modulus at constant stress $E_c(t)$:

$$E_c(t) = \frac{\sigma}{\epsilon(t)}$$



Anelasticity (Viscoelastic behaviour)

- Behaviour of most polymers is in **between** behavior of elastic and viscous materials.
 - At low temperature and high strain rate, polymer demonstrate **elastic behavior**.
 - At high temperature and low strain rate, polymer demonstrate **viscous behavior**.
 - At intermediate temperature and strain rate, polymer demonstrate **viscos-elastic behavior**.
- Instantaneously elastic strain followed by viscous time depended strain.



$$\sigma = \eta \frac{de}{dt}$$

Anelasticity (Viscoelastic behaviour)

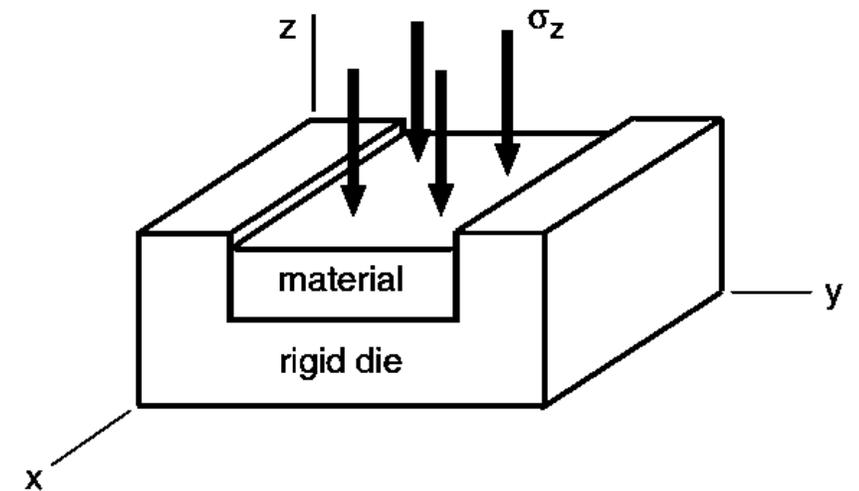
- If the load is removed after a time t_0 , the strain decreases instantaneously by the time-independent strain ε_0 and then reduces slowly to zero, which is called **intermediate viscoelastic behavior**.
- The **retardation time** τ_{ret} is defined as the time needed to reduce the time-dependent part of the strain by a factor $1/e$.
- For **totally viscous behaviour**, deformation or strain is not instantaneous; in response to an applied stress, deformation is delayed or dependent on time (see the figure here).
- This deformation is **not reversible** or completely recovered after the stress is released.



Example:

A sample of material subjected to a compressive stress σ_z is confined so that it cannot deform in the y-direction, as shown in the figure. Assume that there is no friction against the die, so that deformation can freely occur in the x-direction. Assume further that the material is isotropic and exhibits linear-elastic behavior. Determine the following in terms of σ_z and the elastic constants of the material:

- The stress that develops in the y-direction.
- The strain in the z-direction.
- The strain in the x-direction.
- The stiffness $E' = \sigma_z / \epsilon_z$ in the z-direction. Is this apparent modulus equal to the elastic modulus E from a uniaxial test on the material? Why or why not?



- Assume that the compressive stress in the z-direction has a magnitude of 75MPa and that the block is made of a copper alloy, and then calculate σ_y , ϵ_z , ϵ_x and E' .

Solution:

The situation posed requires substituting $\epsilon_y = 0$ and $\sigma_x = 0$, and also treating σ_z as a known quantity.

a. The stress in the y-direction is obtained from:

$$0 = \frac{1}{E} [\sigma_y - \nu(0 + \sigma_z)], \quad \sigma_y = \nu\sigma_z$$

b. The strain in the z-direction is given by using this σ_y

$$\epsilon_z = \frac{1}{E} [\sigma_z - \nu(0 + \nu\sigma_z)], \quad \epsilon_z = \frac{1 - \nu^2}{E} \sigma_z$$

c. The strain in the x-direction is given by using the value of σ_y obtained in (a)

$$\epsilon_x = \frac{1}{E} [0 - \nu(\nu\sigma_z + \sigma_z)], \quad \epsilon_x = -\frac{\nu(1 + \nu)}{E} \sigma_z$$

d. The apparent stiffness in the z-direction is obtained immediately from the equation for ϵ_z

$$E' = \frac{\sigma_z}{\epsilon_z} = \frac{E}{1 - \nu^2}$$

Solution:

For the copper alloy, Table 5.2 provides constants, $E=130$ GPa, and $\nu=0.343$. The compressive stress requires that a negative sign be applied, so that $\sigma_z = -75$ MPa. Substituting these quantities into the equations previously derived gives

$$\sigma_y = \nu\sigma_z = (0.343)(-75 \text{ MPa}) = -25.7 \text{ MPa}$$

$$\varepsilon_z = \frac{1 - \nu^2}{E} \sigma_z = \frac{1 - 0.343^2}{130000} (-75) = -509 \times 10^{-6}$$

$$\varepsilon_z = -\frac{\nu(1 + \nu)}{E} \sigma_z = -\frac{0.343(1 + 0.343)}{130000} (-75) = 266 \times 10^{-6}$$

$$E' = \frac{E}{1 - \nu^2} = \frac{130000}{1 - 0.343^2} = 147300 \text{ MPa}$$